



Turbo Pumps From Theory to Practice

Andreas Schopphoff

Pfeiffer Vacuum GmbH

35614 Asslar / Germany

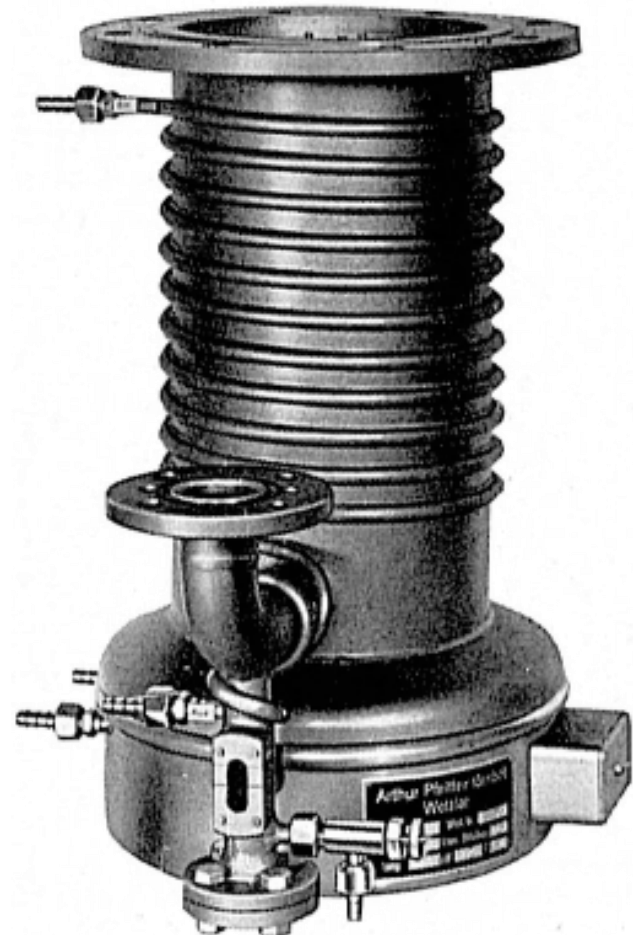
andreas.schopphoff@pfeiffer-vacuum.de

Agenda

- History of the development of Turbo Pumps
- Theory of pumping effect
- Different Design Concepts
- Bearing concepts
- Gas type depending pump performance
- Calculation of the chamber pressure
- Application: turbo pumps in radioactive environment
- Summary

History of High Vacuum

- It was possible to create high- and ultrahigh vacuum with diffusion pumps only
- High energy consumption
- Problem of oil contamination of the vacuum



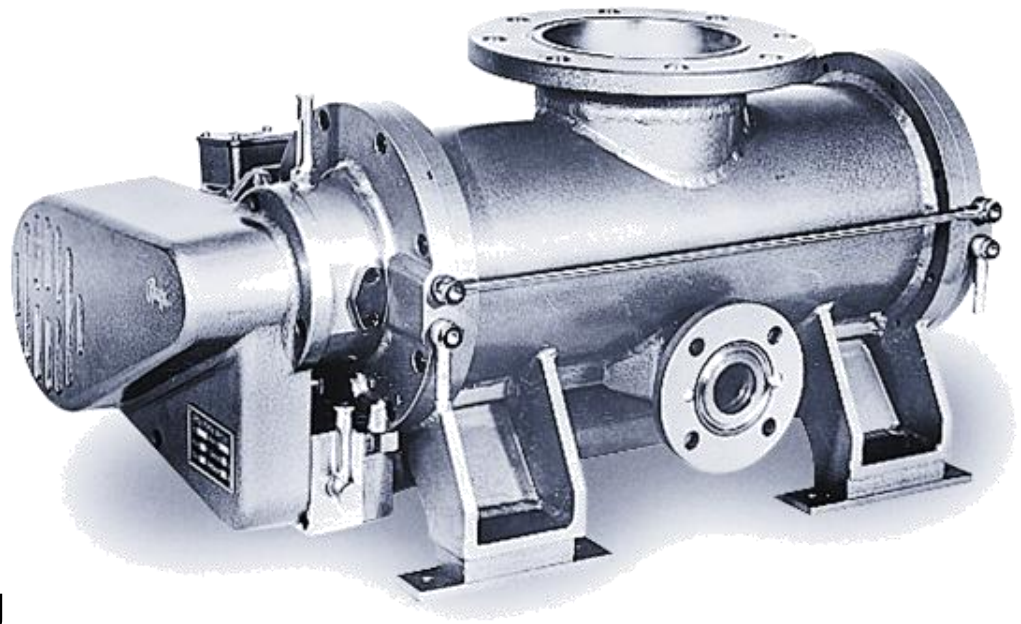
► Invention of the Turbo Pump

- In 1958 Dr. BeckerPfeiffer Vacuum invents the worlds first Turbo Pump
- Invention was done by coincidence

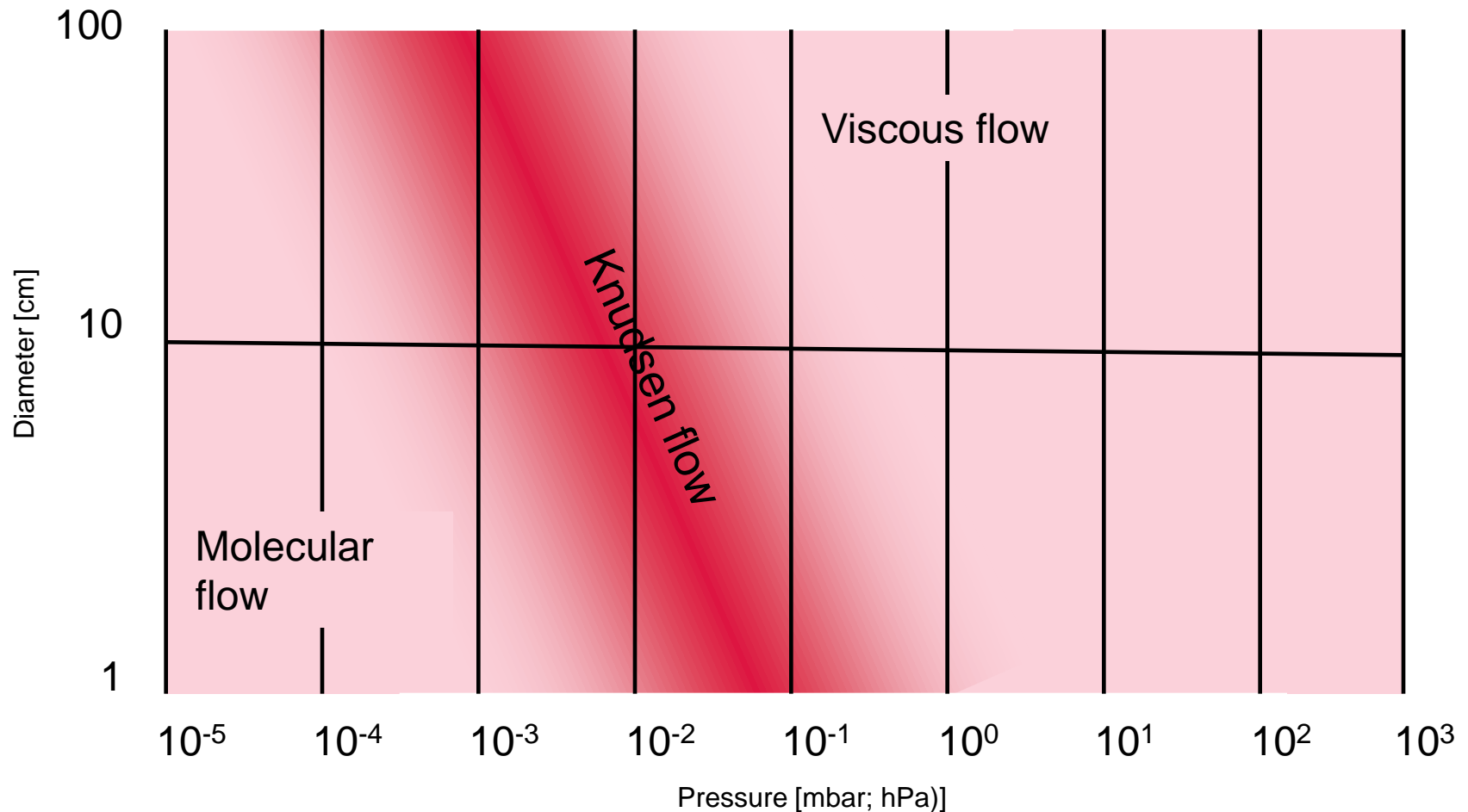
TVF 400

Pumping speed
140 l/s for N₂

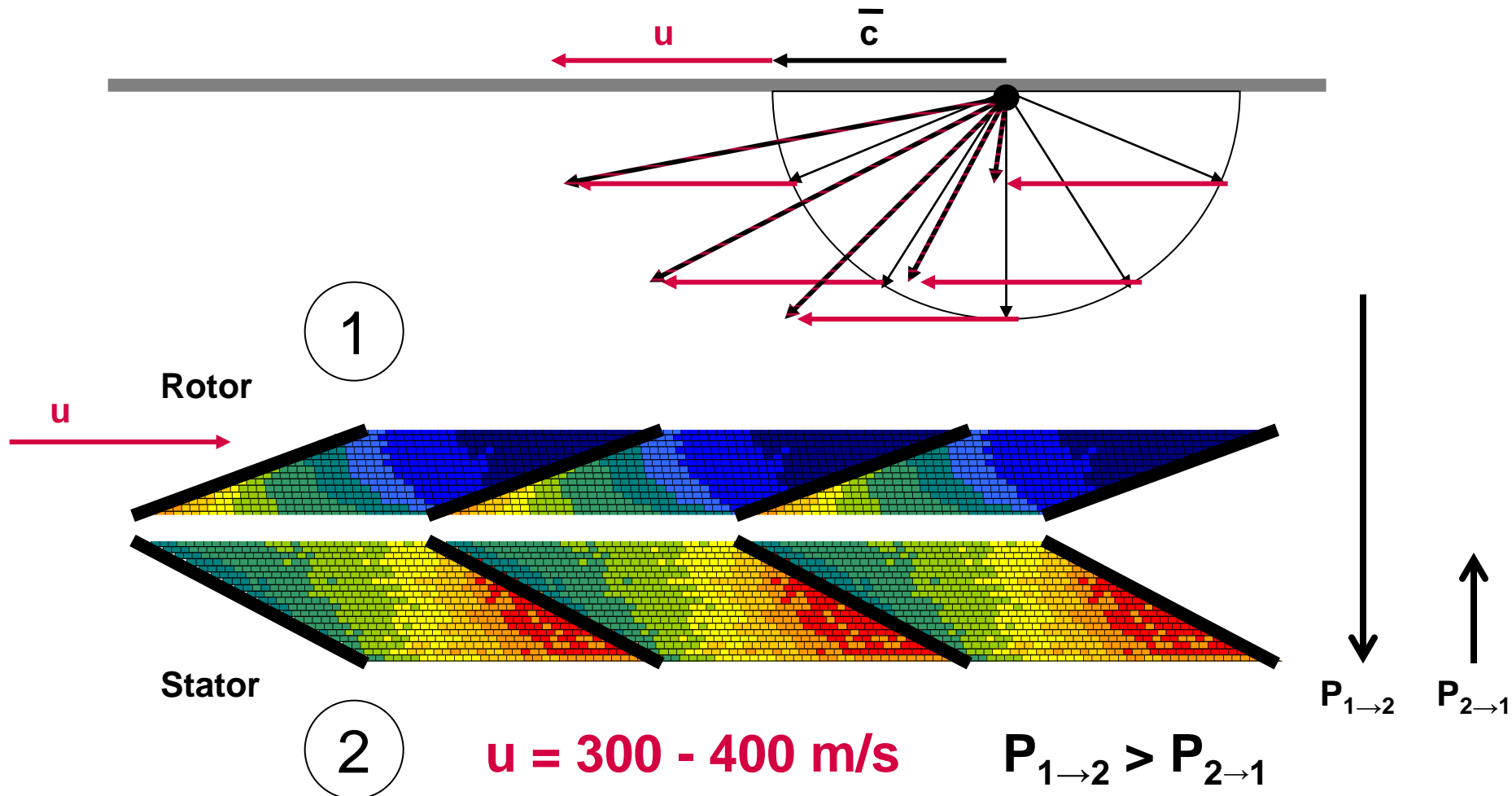
Weight: approx. 60 Kg



Different Flow Regimes

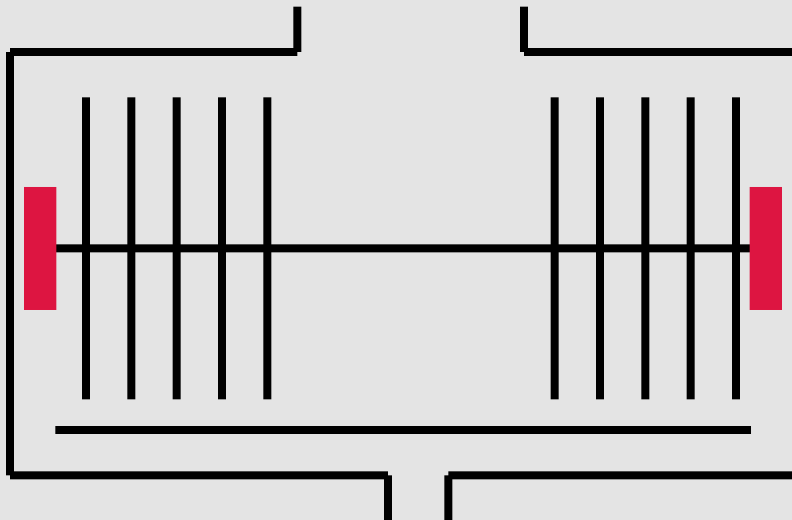


Theory of Turbo Pumping Effect



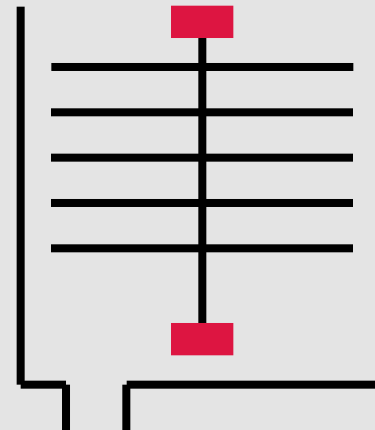
Turbo Pump Design

Classical Design



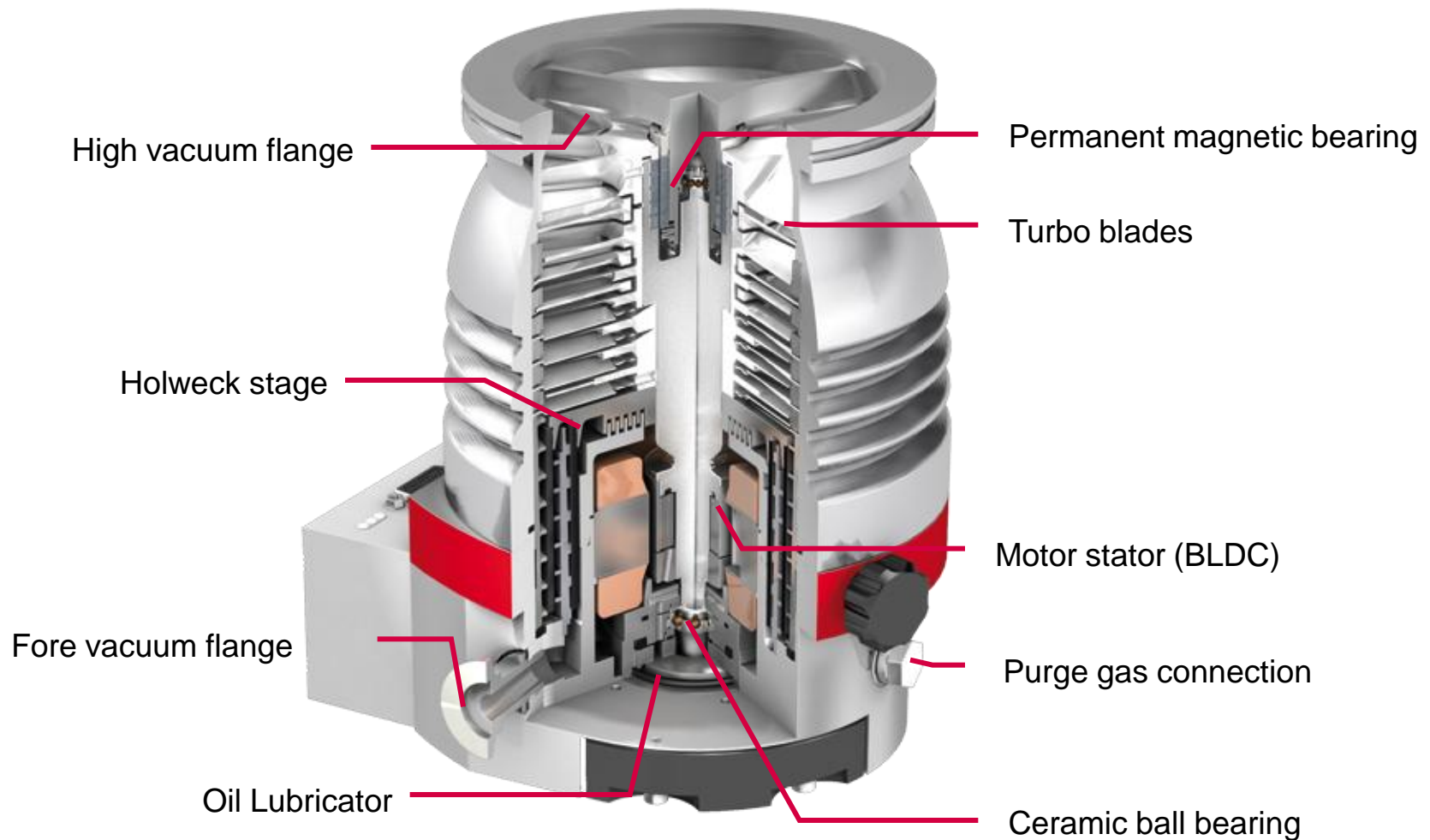
Double flow design

Modern Design

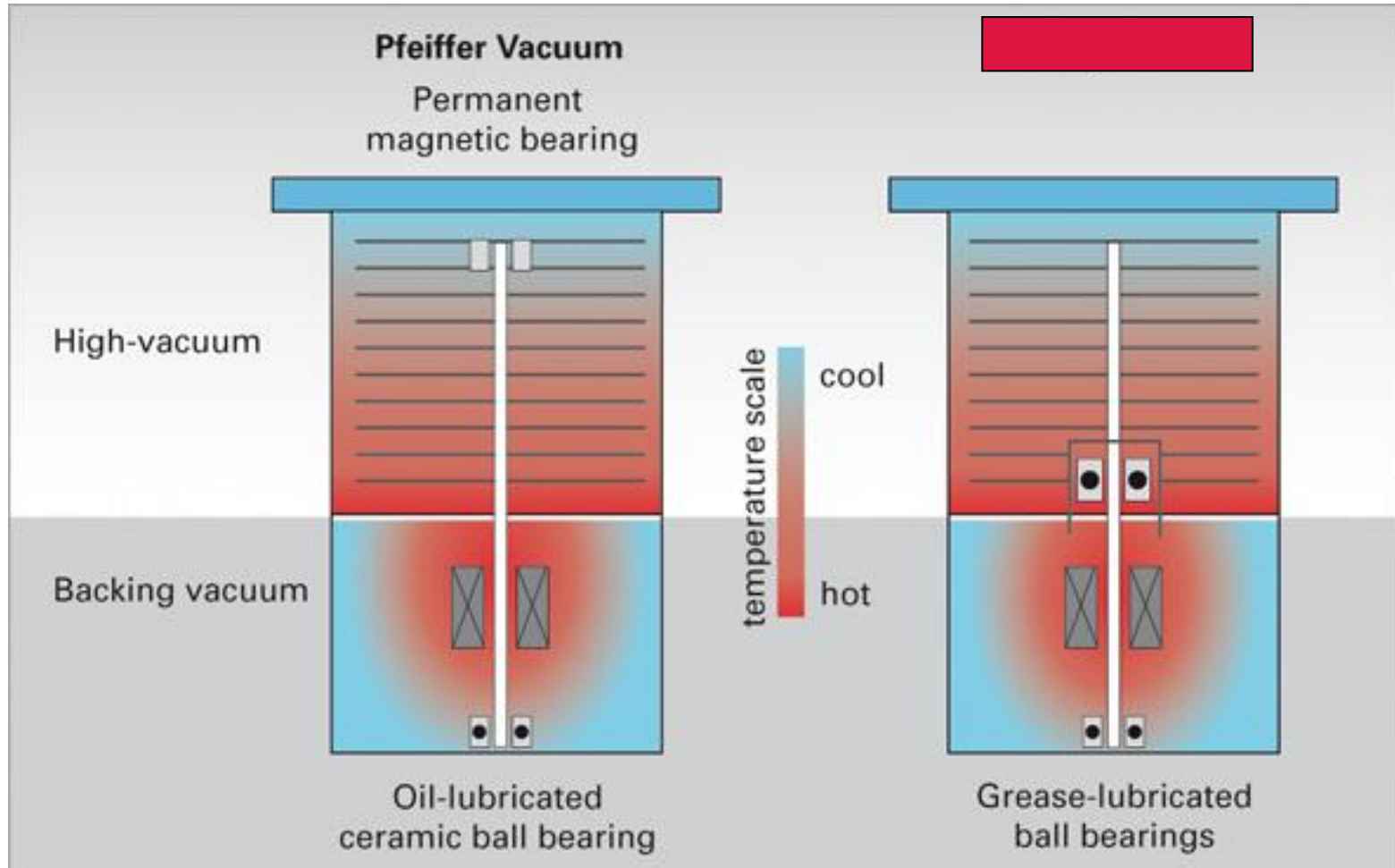


Single flow design

Turbo Pump Design



Bearing Types and Rotor Design



Turbo Pumps today

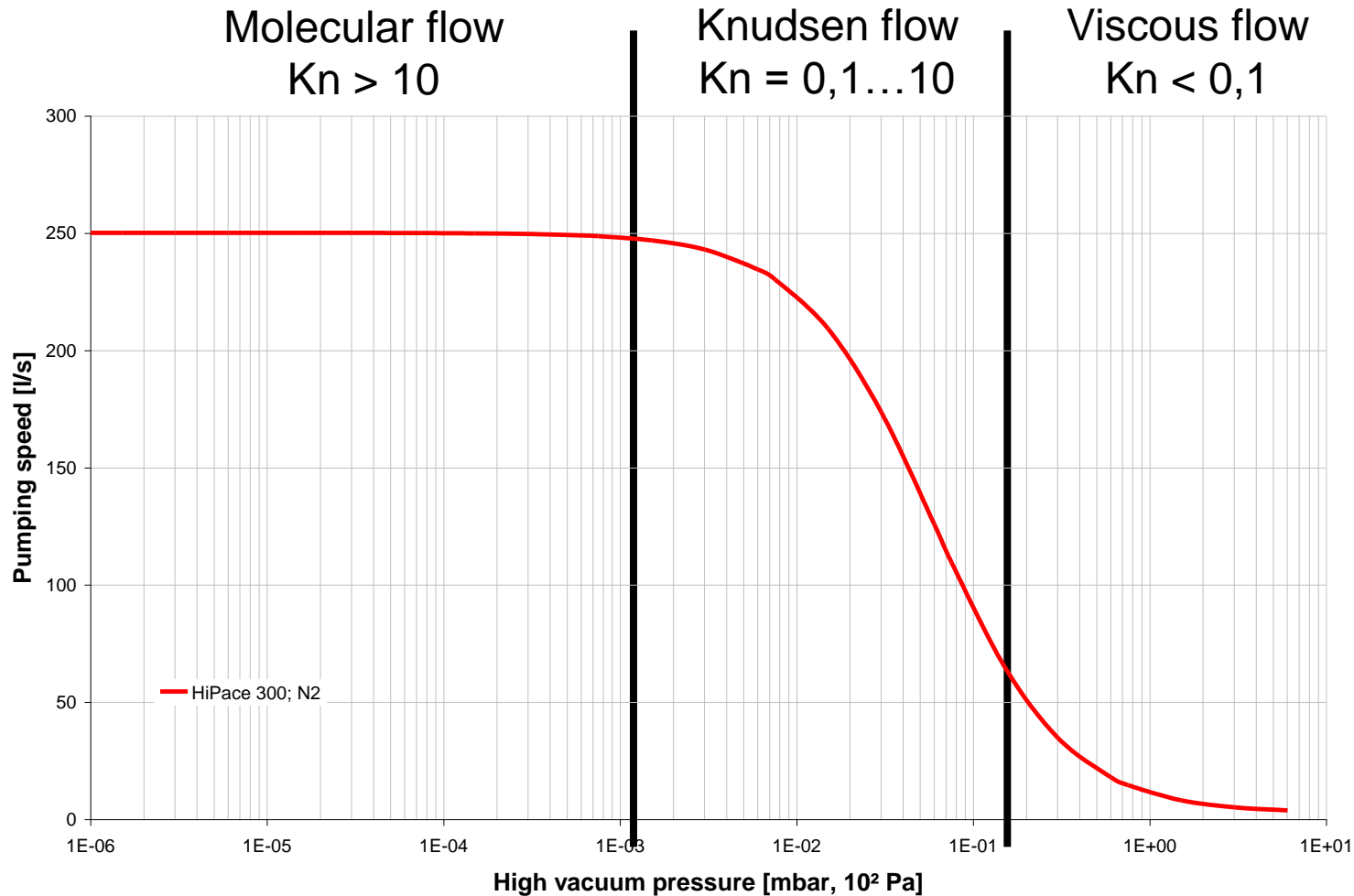
HiPace 300

Pumping speed
260 l/s for N₂

Weight: 6.2 Kg



Pumping Speed vs. Regimes



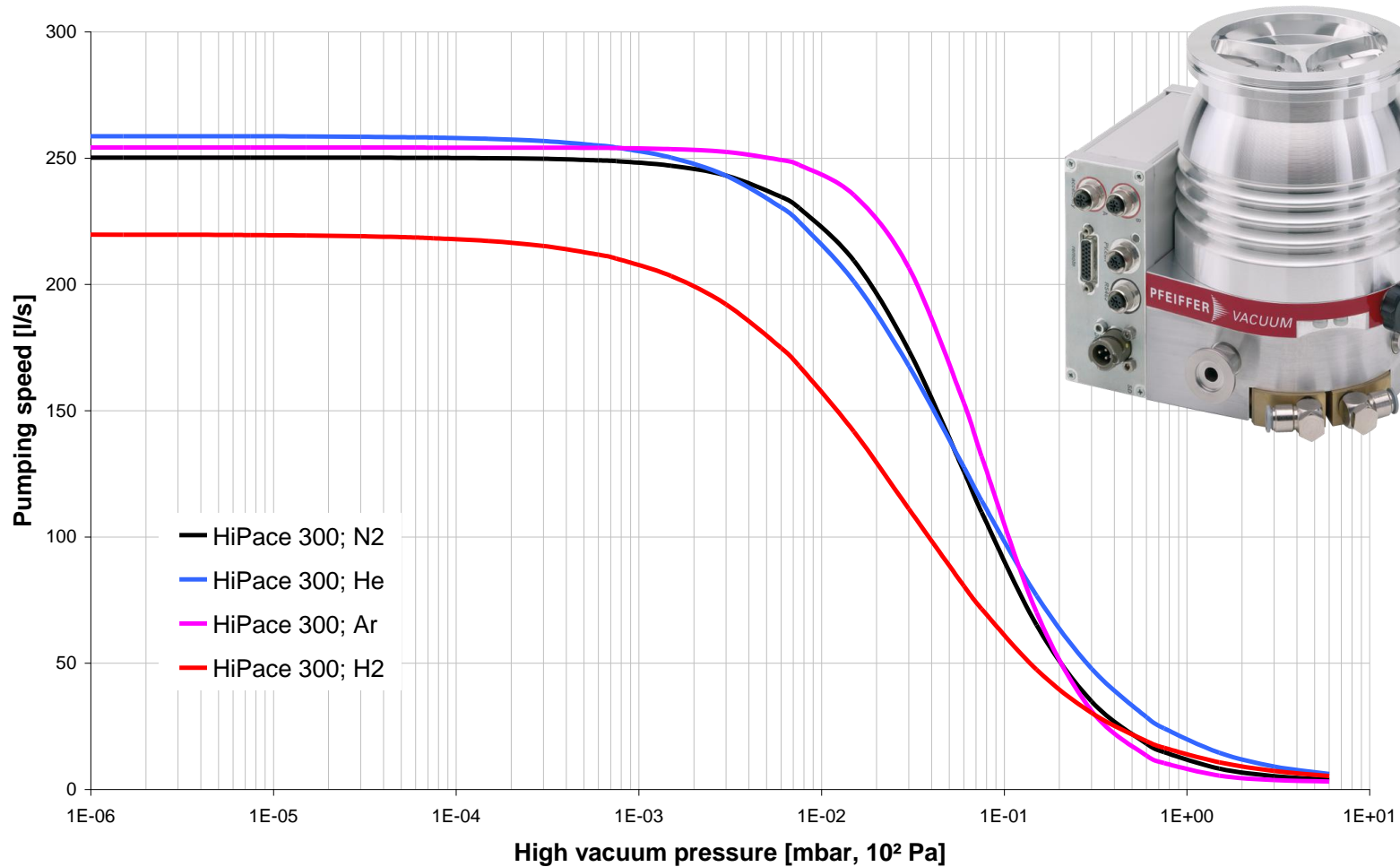


Important Technical Data

- Pumping speed S [l/s]
- Compression ratio ($Q=0$) : $K_0 = p_{FV} / p_{HV}$
- Max. backing pressure: $p_{FV \max.}$ [mbar]
- Max. gasload: $Q_{\max.}$ [mbar l / s]

(according to ISO 5302: Vacuum Technology TMP-Measurement and Performance)

Gas Type dependant Pumping Speed





Technical Data

Example HiPace 300

- Pumping speed

$$S_{N_2} = 260 \text{ l/s}$$

$$S_{H_2} = 220 \text{ l/s}$$

- Compression ratio (Q=0) :

$$K_{0 N_2} = > 1 \text{ E } 11$$

$$K_{0 H_2} = 9 \text{ E } 5$$

- Max. backing pressure:

$$p_{FV \text{ max.}} = 20 \text{ mbar l/s}$$

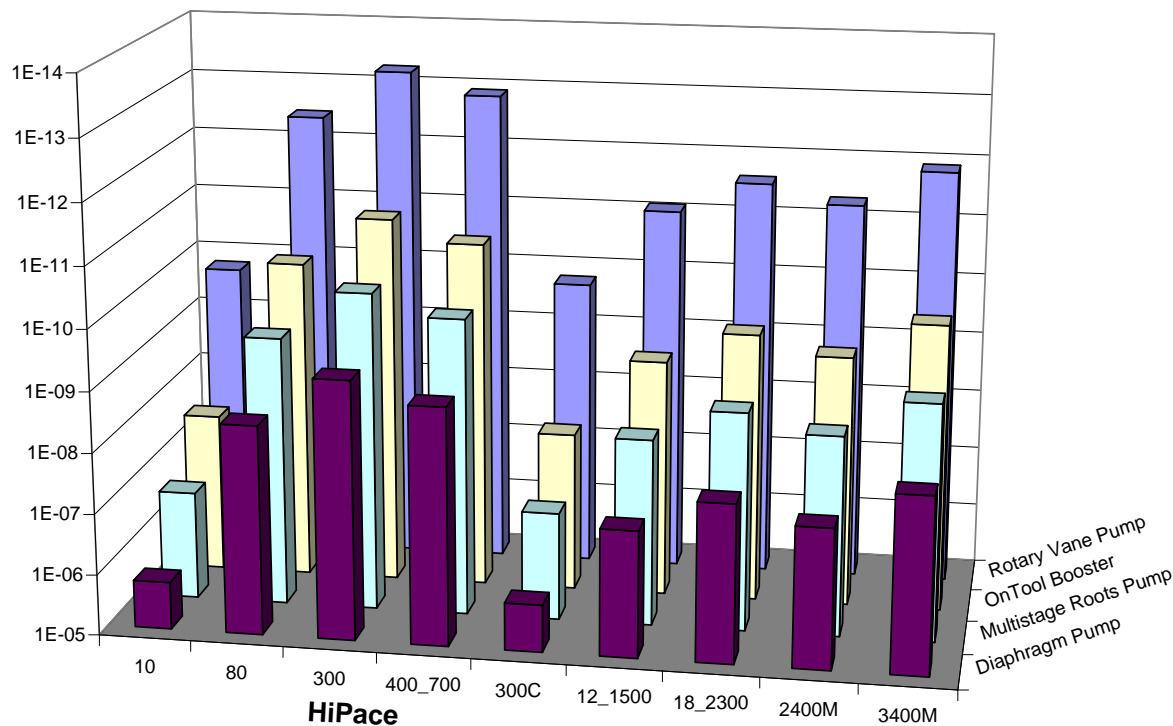
- Max. gas load:

$$Q_{\text{max.}} = 14 \text{ mbar l/s}$$

(according to ISO 5302: Vacuum Technology
TMP-Measurement and Performance)

Backing Pump Selection for UHV

(U)HV - partial
pressure H_2 in
mbar



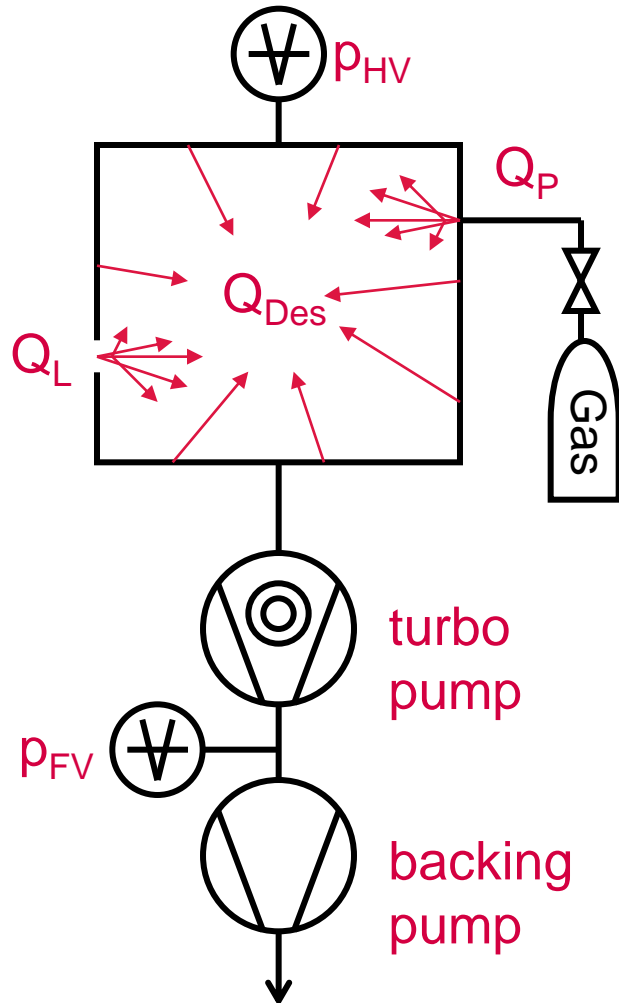
(w/o desorption effects)

$$p_{HVi} = \frac{p_{Atm i}}{k_{oiBP} \cdot k_{oiTMP}}$$

$$p_{HVi} = \frac{5 \cdot 10^{-3} \text{ mbar}_{H_2}}{1 \cdot 10^6_{RVP} \cdot 1 \cdot 10^4_{TMP}}$$

$$p_{HVi} = 5 \cdot 10^{-13} \text{ mbar}_{H_2}$$

Chamber Pressure Calculation

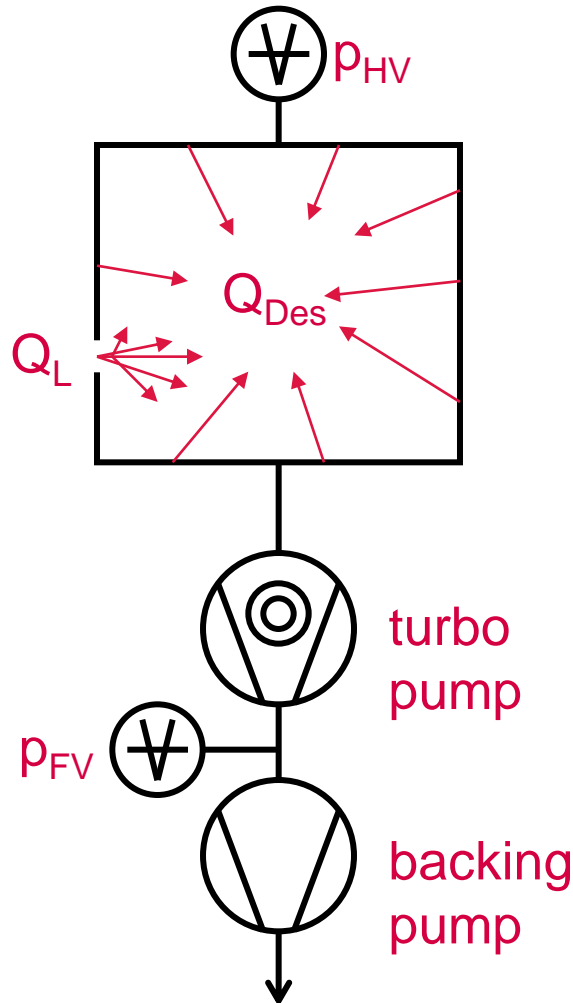


Gas Load Applications

$$p_{HV} = \frac{Q_P + Q_{Des} + Q_L}{S_{TMP,eff}} + \frac{p_{FV}H_2}{K_{0H_2}} + \frac{p_{FV}G_2}{K_{0G_2}} + \dots$$

- Q_P = Process gas flow
- Q_{Des} = Desorption gas flow
- Q_L = Gas flow caused by leaks

Chamber Pressure Calculation



UHV Applications

$$p_{HV} = \frac{Q_{Des} + Q_L}{S_{TMP,eff}} + \frac{p_{FVH_2}}{K_{0H_2}} + \frac{p_{FVG2}}{K_{0G2}} + \dots$$

Q_{Des} = Desorption gas flow

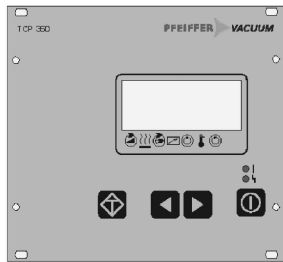
Q_L = Gas flow caused by leaks

K_0 = Compression ratio

p_{FV} = Fore vacuum pressure

Turbo Pumps with external Controller

Elektronics TCP 350



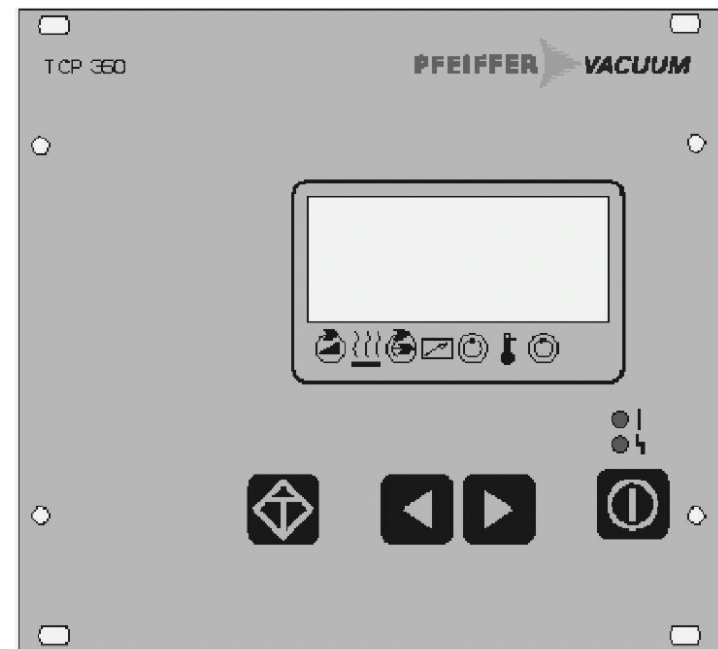
Turbo Pump



Up to 1000m

TCP 350 - Controller for radiation environment

- Turbo controller for rack mounting
- For applications in radiation environment
- No semiconductor elements on the pump



Easy on Side Service

- Bearing installed in a cartridge
- Easy to exchange
- Exchange possible on side
- No adjustment of the bearing
- No rotor rebalancing required
- Service interval 4 years in clean applications

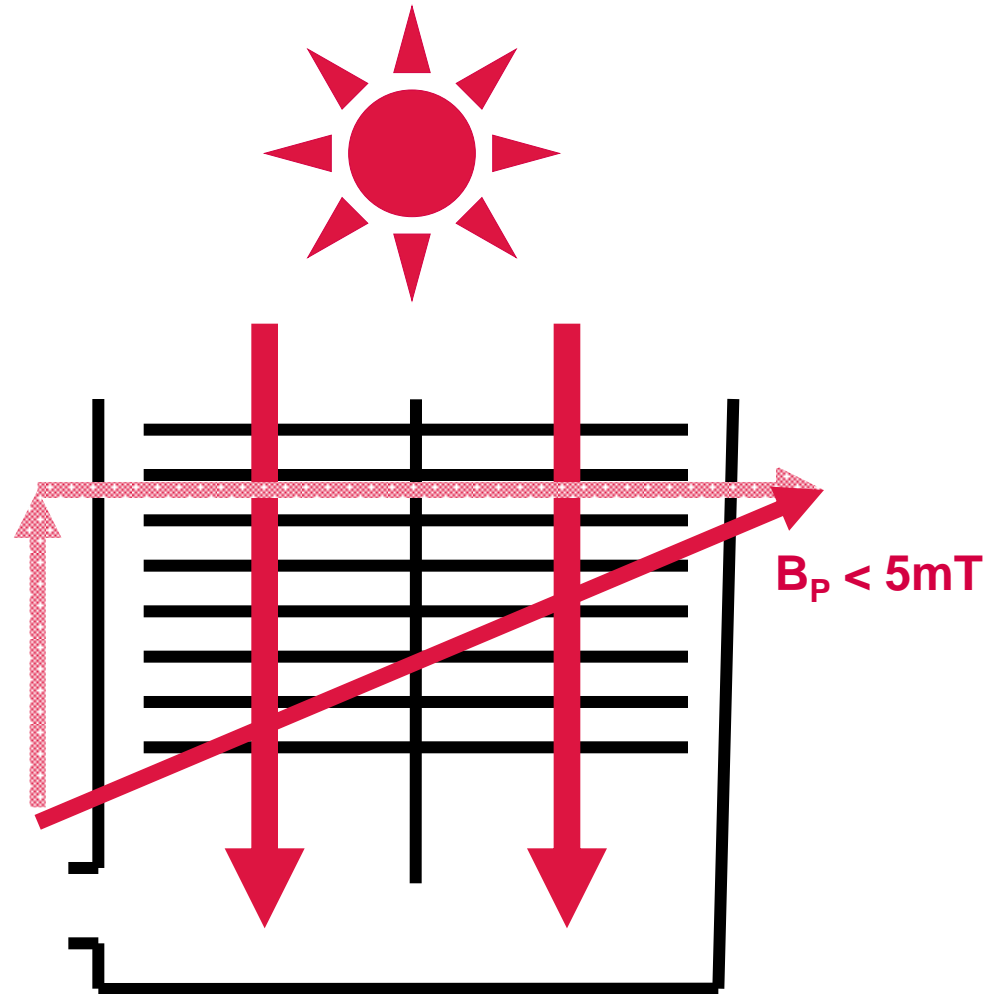




Application Limits

Rotor Temperature < 120°C

- Gas friction
- Magnetic fields
- Heat radiation



Summary

- The turbo pump is suitable for very many applications
- It has been proven as a reliable tool in all kind of industries
- With modern technologies it is easy to operate and to maintain
- Even for UHV applications it is the right choice



Turbopumps from Theory to Practice

Andreas Schopphoff

Pfeiffer Vacuum GmbH

35614 Asslar / Germany

andreas.schopphoff@pfeiffer-vacuum.de